Experimental investigation of dynamic impact of roadheader on floor

This article presents the selected results of experimental examinations performed on the test stand in the Technological Hall of the Faculty of Mining and Geology at the Silesian University of Technology in which an R-130 roadheader (by Famur S.A.) is installed. During the examinations, the courses of dynamic load were determined in the points of the roadheader’s boom support during its operation. In order to determine the impact of the cutting process on the strengths transferred onto the floor, the distribution of the static load of the roadheader’s supports caused by its own weight with its changing center of gravity caused by the boom deflecting were determined. During the examinations, the size and nature of the dynamic impact of the roadheader on the floor in its support points while cutting the surface of the block made of equivalent materials (cement and sand masses) of various uniaxial compressive strength (UCS) was determined experimentally. The nature of the roadheader’s chassis vibrations caused by cutting as well as the changeability of the location of its anchorage point with the movement of the cutting heads on the surface being mined result in high load fitfully in the roadheader’s support points. The dynamic impact of the roadheader on the floor also has a strongly dynamic character.

Key words: roadheader, cutting process, dynamic load, floor reaction, experimental investigations

1. INTRODUCTION

Numerous observations of the operation of boom-type roadheaders carried out under service conditions and simulation examinations performed in various scientific centers show that the cutting process of especially rocks of low workability (hard rocks) is a source of strongly enforced vibrations and dynamic loads [1–5]. These are present mostly in the roadheader’s subassemblies, including the cutting system in the form of a boom with the cutting head’s drive installed on it. They are, however, transferred further on through structural joints to the turntable, the roadheader’s main frame, and other of the roadheader’s subassemblies connected with it. The vibrations generated by the cutting process by the roadheader’s supports are transferred to the floor of the tunnel.

The component of the vibrations of the roadheader’s body perpendicular to the floor is especially important from the point of view of the dynamic impact of the roadheader on the rock mass surrounding the tunnel. The high intensity of the vibrations that may lead to even a momentary loss of contact with the floor by the roadheader is a source of strong loads of an impact character. These may be the reason for the shaking of a para-seismic nature. The types of vibrations caused by human activity (including, for example, mining extraction or of a transportation origin) have a negative impact on the environment as well as the population [6–8]. The dynamic strengths of the roadheader’s impact on the floor may then be the reason for its degradation as a result of exceeding its uniaxial compressive strength (UCS) or resistance to surface pressure. The effect is especially undesirable in tunnels with long service lives, as this leads to
the necessity of floor reconstruction in order to adjust its condition to the requirements resulting from the function of the tunnels (e.g. transportational).

The roadheader is not rigidly fixed in the tunnel, so it is the frictional contact between its supports and the floor that essentially decides about maintaining stability. The contact results from the value of the friction coefficient between the surfaces that are in contact and with the roadheader’s pressing force to the floor in the place of its support. The support of the roadheader on the floor when mining the surface of the heading face is performed with the use of a loading device – from the front and bracing jack(s) at the back of the roadheader. In theory, we then have the linear support along the front edge of the loading device and point support on the bracing jack. In reality, however, the loading table may also have a contact point with the floor as a result of floor irregularity or a rock mass present on it. The roadheader’s body vibrations that are perpendicular to the floor surface may be the reason for the significant reduction of the frictional contact strength values and even the loss of adhesion to the floor by the roadheader in its support points. As a consequence, it will lead to a reduction or even loss of the possibility of balancing the forces that act on the roadheader’s body on the plane that is parallel to the floor from mining and loss of stability by the roadheader (its uncontrolled movement along the floor surface). If this is the case, an efficient realization of the cutting process will not be possible.

Taking into account the possibility of the effective and safe mining of rocks (especially those of low workability using a boom-type roadheader), it is really important to determine the size and character of the dynamic impact of the types of machines on the floor. One of the ways of identifying the impact is to measure the reaction forces acting on the floor during the cutting process. From a technical point of view, the task is not easy to perform, especially when the measurement takes place under service conditions (in the underground roadway of a mine or while drilling a tunnel). However, the easy-to-use measuring systems that allow for the measuring of the normal floor reaction reduced to force as well as matrix sensors (measuring mats) are already known. The first ones are commonly applied to the needs of controlling the vehicle load in the road and railway transportation [9, 10]. In turn, the measuring mats allow us to measure the pressures [11]. In the examinations on self-propelled machines, they may be used for the purposes of determining the real distribution of the machine chassis pressure on the floor. The solutions, however, are not suitable for examinations on the dynamic impact of the roadheader on the floor, as they do not allow us to measure reaction in the case of losing contact with the floor by the machine.

The measurements of the dynamic impact of the boom-type roadheader on the floor were carried out on the test stand in the Technological Hall of the Faculty of Mining and Geology at the Silesian University of Technology. They were performed while cutting on a block made of equivalent materials (cement and sand masses of various UCS) using an R-130 roadheader made by Famur S.A. In order to do this, the roadheader was placed on four supports constructed especially for it [12] – Figure 1. In each support, four tensometric single-axis (compression) force sensors have been built-in: two for measuring

![Fig. 1. Method of locating R-130 roadheader at test stand on four supports (a) and view of one support equipped with four tensometric force sensors (b) [12]](image-url)
the forces on the plane parallel to the floor (longitudinal and transverse reaction), and two for measuring the reaction perpendicular to the floor. The application of two force sensors in the direction perpendicular to the floor results from the possible return of the reaction component during the cutting process. Depending on the distribution of forces acting on the roadheader, it may be pushed in the given support point to the floor or lifted upwards. The measuring system applied during the experimental examinations allows us to determine the value of force necessary to ensure a balance in each of the roadheader’s support points, even in the cases when the roadheader may lose its contact with the floor under real conditions.

The article presents the selected results of our experimental examinations. The reaction components in the roadheader’s support points acting in the directions perpendicular to the floor have been analyzed.

2. ROADHEADER’S WEIGHT LOAD ANALYSIS

In order to identify the distribution of reactions transferred onto the floor, the roadheader was placed on a system of supports located in pairs on each side of it (Fig. 2).

The analysis of the dynamic impact of the roadheader’s chassis on the floor at the support points was carried out based on a series of measurements for various boom positions while cutting on the cement and sand block. Because of the change of the boom location during the cutting process, an analysis of the impact of the boom position on the distribution of the support loads perpendicular to the floor (Z axis) from the roadheader’s weight was carried out first. The analysis was based on the measurements of the pressure value on the given supports with various settings of the roadheader’s boom’s deflecting angle in vertical plane $\alpha_V$ (perpendicular to the floor) and in the full range of the boom deflecting angle on the plane parallel to floor $\alpha_H$.

In order to determine the reference level, the courses of the forces in the supports without cutting (from the roadheader’s weight) were registered. The courses of the loads in the supports towards the Z axis depending on the boom location that influences the location of the roadheader’s center of gravity were determined. When deflecting the boom, the roadheader’s center of gravity changes because of the movement of the boom and the moving part of the roadheader’s turntable. Therefore, the values of the forces acting on the given supports with the boom

![Diagram of support system location and reaction directions on force sensors: PL – front-left support; PP – front-right support; TL – back-left support; TP – back-right support](image)
deflecting on the plane parallel to the floor (Fig. 3) change. The front supports (PP and PL) are the most loaded. Because of the location of the roadheader’s center of gravity, their values change the most while deflecting the boom.

The four supports of the examined roadheader make up a system that is statically indeterminate; this is why the distribution of loads perpendicular to the floor on the given supports depends on the supports’ performance, floor flatness, accuracy of settings of the pressure sensors, and rigidity of the roadheader’s support system. In order to eliminate the impact of all of these factors, the sum of the loads in the front (PP+PL) and back supports (TP+TL) was determined, which allows for stating the location of the roadheader’s center of gravity in relation to the front supports. The sum of the loads in the right (PP+TP) and left supports (PL+TL) was also determined, which allows for stating the distance of the center of gravity from the roadheader’s longitudinal axis (Fig. 4).

The sum of the loads registered in the front (PP+PL) and back supports (TP+TL) as well as the sum of the loads registered in the right (PP+TP) and left supports (PL+TL) within the whole range of the boom deflecting on the plane parallel to the floor $-35^\circ \leq \alpha_H \leq +35^\circ$ (Fig. 5) for two locations of the boom on the plane perpendicular to the floor $\alpha_V = 0^\circ$ (boom located horizontally) and $\alpha_V = +16^\circ$ (boom lifted upwards) are shown in Figure 4. For the boom set horizontally along the roadheader’s axis ($\alpha_V = 0^\circ$; $\alpha_H = 0^\circ$), the cumulative load of the roadheader’s front supports (PP+PL) is 232 kN, which makes up 88% of the whole roadheader’s weight. The roadheader’s center of gravity determined in the way referred to in [13] is located in this case at

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**Fig. 3. Support load caused by roadheader’s weight in direction perpendicular to floor (towards Z axis)**

**Fig. 4. Sum of registered loads of front (PP+PL) and back supports (TP+TL) and right (PP+TP) and left supports (PL+TL) caused by roadheader’s weight for two boom deflecting angle values in plane perpendicular to floor $\alpha_V$**
a distance of 224 mm behind the front supports and 116 mm in front of the roadheader’s turntable axis of rotation. The center of gravity is shifted 22 mm to the right from the roadheader’s longitudinal axis as for the analyzed location of the boom. The cumulative load of the right supports (PP+TP) amounting to 134 kN makes up 51\%/G25\% of the roadheader’s weight.

Deflecting the boom on the plane parallel to the floor from the roadheader’s longitudinal axis leftwards and rightwards causes a reduction in the load of the front supports and an increase in the load of the back supports, which is the result of the dislocation of the roadheader’s center of gravity towards the back supports. When deflecting the boom left, the load of the left supports significantly increases and that of right supports decreases. And so, for $\alpha_H = -30^\circ$ and $\alpha_V = 0^\circ$, the load of the front supports (PP+PL) decreases by ca. 6 kN, which corresponds to the shifting of the roadheader’s center of gravity backwards by 50 mm. The load of the left supports (PL+TL) increases by 22 kN, which results in the shifting of the center of gravity leftwards by 194 mm.

Lifting the boom upwards with an angle of $\alpha_V = +16^\circ$ moves the center of gravity towards the roadheader’s rear, which causes a slight mitigation in the front supports and loading of the back supports of the roadheader. It does not, however, have a visible impact on the cumulative load of the right and left supports (Fig. 4).

### 3. ANALYSIS OF DYNAMIC LOAD CAUSED BY CUTTING IN DIRECTION PERPENDICULAR TO FLOOR

During the experimental examinations performed on the test stand, the courses of the dynamic load have been registered in the points of the roadheader’s boom support during its operation. The dynamic impact of the roadheader on the floor generated by the cutting process while performing its upper and lower cuts was analyzed. The roadheader mines the soil in cuts that are parallel to the floor by means of moving the boom without changing the location of its chassis. Boom lifting or lowering after the cut at the same time determines the height of the subsequent cut. If the boom is lifted, then the next cut is an upper one, and if the boom is lowered, then the next cut is a lower cut. In the case of transverse cutting heads that work undershot while performing upper cuts, the picks perform a degressive cut (with decreasing cut depth – Fig. 5a) and while performing lower cuts, the picks cut progressively (with the increasing cut depth – Fig. 5b). And so, in both cases, the way of loading the transverse cutting heads with cutting forces is different, which influences the method of loading the roadheader’s chassis in a direction perpendicular to the floor.

The cutting process is a source of strongly enforced vibrations and dynamic loads that, through
the structural joints, are transferred further onto the turntable, the main frame of the roadheader, and other roadheader subassemblies connected with it. The vibrations generated by the cutting process by the roadheader’s supports are transferred to the floor. During the examinations on the test stand while cutting on the cement and sand block using the R-130 roadheader, the forces (which are of a significant importance) transferred on the roadheader’s supports perpendicular to the floor were measured and registered. Two performed cuts were analyzed when the boom was deflecting rightwards from the left side; the first cut was an upper cut, while the other was a lower cut.

With a height of 107 mm, the upper cut was performed with the boom located on the plane perpendicular to the floor $\alpha_V = +7^\circ$ during its deflecting on the plane parallel to the floor within a range of angle $\alpha_H$ from $-22^\circ$ to $+12^\circ$ in 250 seconds with 10-second intervals (Fig. 6).

Block cutting by the transverse cutting heads of the roadheader had a significant impact on the values and character of the forces perpendicular to the floor transferred in the roadheader’s support points. The mean cumulative values of the loads of the front (PP+PL) and back supports (TP+TL) change less as opposed to the cumulative mean loads of the front (PP+TP) and left supports (PL+TL) on the courses in which the change of the location of the roadheader’s center of gravity is clearly marked as the boom deflecting (Fig. 6).

In order to determine the impact of the cutting process on the forces transferred onto the floor from the values of the loads measured in the supports during the cutting process, the values of the loads caused by the roadheader’s weight have been deducted. Using the registered time functions of the change of the boom deflecting angle value on the plane parallel to the floor and previously specified dependencies of the cumulative loads of the front and back supports of the roadheader in the angle function, the time functions of the support loads resulting from only the cutting process were obtained (Fig. 7). The cutting forces acting on the picks located on the transverse cutting heads when performing an upper cut have a significant impact on the increase of the cumulative load of the back supports and slight mitigation on the front supports. The mean value of the back support load and front support mitigation depends on torque $M_M$ of the cutting system drive (course shown as orange in Figure 7). The load cumulative courses for both the front and back supports are characterized by high changeability, and their amplitude (understood as the difference between the maximum and minimum value) exceeds 40 kN.

Mining causes a significant increase in the value of the load acting perpendicularly to the floor in the supports located on the left side of the roadheader (PL+TL) as compared to the supports on the right side (PP+TP), which is the result of the boom’s torque during its deflecting from the left side rightwards (Fig. 8). The courses of the cumulative forces that load the left and right supports are characterized by lower changeability than it is in the front and back supports. The reason of the differentiation of the force vibration amplitude between the front and back supports is illustrated in Figure 8.
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[supports (PP+PL and TP+TL in Figure 7) and the right and left supports (PP+TP and PL+TL in Figure 8)] is the cumulation of forces in the given supports. For the measurement fragment between the 150th and 160th second of the measurement, the time functions of the loads perpendicular to the floor have been presented separately for all supports (Fig. 9). They are characterized by a clear repeatability of a period equal to the time of the cutting head’s rotation and similar changeability range. The forces in the front-right (PP) and front-left supports (PL) change in the same way. The courses are more or less a mirror reflection of the course of the cutting head’s drive engine torque (momentary torque increase causes a momentary mitigation of the front supports). Meanwhile, the vibrations of the forces in the back-right (TP) and back-left supports (TL) have the same character as the course of the cutting head’s drive engine torque (a momentary torque increase causes a momentary load of the back supports). This means that the vibrations of the forces in the front and back supports are mirror images, which causes that the amplitudes of the vibrations increase when cumulating the loads in the front (PP+PL) and back supports (TP+TL) while the amplitudes of the vibrations decrease when cumulating the loads in the right (PP+TP) and the left supports (PL+TL).
With a height of 121 mm, the lower cut was performed with the boom located on the plane perpendicular to the floor $\alpha_V = -15^\circ$ during its deflecting on the plane parallel to the floor within the range of angles $\alpha_H$ from $-30^\circ$ to $+14^\circ$ in 55 seconds (Fig. 10). The performance of the lower cut using the transverse cutting heads of the roadheader had a significant impact on the values and character of the forces perpendicular to the floor transferred in the roadheader’s support points. The mean values of the cumulative loads of the front (PP+PL) and back supports (TP+TL) changed mostly because of the differentiated load of the cutting system. At the same time, the value of the cumulative loads of the back supports (TP+TL), which is almost equal to the value of the cumulative loads of the front supports (PP+PL) in the 55th second of the measurement, significantly increases.

When deducting the values of the loads caused by the roadheader’s weight from the values of the measured loads in the supports when cutting, the forces transferred onto the floor caused by the cutting process were determined. Using the registered time functions of the change of the boom deflecting angle value on the plane parallel to the floor as well as the previously specified dependencies of the cumulative loads of the front and back supports of the roadheader in the angle function, the time functions of the support loads resulting only from the cutting process when performing a lower cut were obtained (Fig. 11).
Just as it was with the upper cut, the cutting forces acting on the picks located on the transverse cutting heads when performing the lower cut influence an increase in the cumulative load of the back supports and mitigation of the front supports. The effect, however, is greater than in the course of the upper cut performance. The cumulative mitigation of the front supports resulting from the cutting process reaches 100 kN in this case (if directed vertically upwards), while for the back supports, it reaches +75 kN. The mean value of the back support load and front support mitigation is proportional here to the value of torque $M_M$ of the cutting system drive (course shown as orange in Figure 11).

Because of the low boom location when performing the lower cut ($\alpha_V = -15^\circ$), the impact of the boom’s torque on the distribution of the reactions perpendicular to the floor on the left and right sides of the roadheader (PL+TL and PP+TP) is small. As a result, it is the cutting vertical reaction component that determines it (Fig. 12).

The vibrations of the cumulative forces that load the left and right supports are characterized by lower changeability than in the front and back supports. This results from the cumulation of the reaction forces from the given supports.

For a ten-second measurement fragment, the time functions of the loads perpendicular to the floor have been presented separately for all supports (Fig. 13).
Just as it was with regard to the upper cut, the vibrations of the forces that load the supports on the plane perpendicular to the floor for the lower cut are characterized by clear repeatability, with the time period equal to the time of the cutting head’s rotation and similar changeability range. The vibrations of the forces in the front-right support (PP) and front-left supports (PL) have the same character and are mirror images of the course of the drive engine’s torque. The vibrations of the forces in the back-right (TP) and back-left supports (TL) also change in the same manner. The courses, however, have the same character as the course of the cutting head’s drive engine’s torque (the momentary torque increases cause a momentary load on the back supports). The mirror image of the courses of the forces in the front and back supports causes an increase in the amplitude of the vibrations in the front (PP+PL) and back supports (TP+TL) and a decrease in the amplitude of the vibrations in the right (PP+TP) and left supports (PL+TL).

In order to identify the distribution of the reactions transferred onto the floor, the roadheader was placed on a system of supports located in pairs on each side of it. The sum of the loads in the front (PP+PL) and back supports (TP+TL) was determined, which allows us to determine the location of the roadheader’s center of gravity in relation to the front supports, and the sum of the loads in the right (PP+TP) and left supports (PL+TL), which allows us to determine the distance of the center of gravity from the roadheader’s longitudinal axis.

The cutting process is a source of strongly enforced vibrations and dynamic loads that, through the structural joints, are transferred further onto the turntable, the main frame of the roadheader, and the other roadheader subassemblies connected with it. The vibrations generated by the cutting process by the roadheader’s supports are transferred onto the floor of the tunnel. The vibrations transferred on the roadheader’s supports perpendicular to the floor for two cuts performed when the boom was deflecting rightwards from the left side were analyzed; the first cut was an upper cut, while the other was a lower cut.

The cutting forces acting on the cutting picks located on the transverse cutting heads when performing an upper cut influence the increase of the cumulative load of the back supports and mitigation of the front supports. The mean value of the back support load and front support mitigation is proportional to the value of the torque of the cutting system drive. In order to determine the impact of the cutting process on the forces transferred onto the floor from the values

3. SUMMARY

The measurements of the dynamic impact of a boom-type roadheader on the floor were carried out on the test stand in the Technological Hall of the Faculty of Mining and Geology at the Silesian University of Technology. They were performed while cutting on a block made of equivalent materials (cement and sand masses of various UCS) using an R-130 roadheader made by Famur S.A.
of the loads measured in the supports during the cutting process, the values of the loads caused by the roadheader’s weight have been separated. The cutting forces acting on the picks located on the transverse cutting heads when performing a lower cut influence the increase of the cumulative load of the back supports and mitigation of the front supports much more than when performing an upper cut.

For both the upper and lower cuts, the vibrations of the support loading forces on the plane perpendicular to the floor are characterized by a similar changeability range and clear repeatability (with a time period equal to the time of the cutting head’s rotation). The vibrations of the forces in both front supports have the same character and are mirror images of the course of the cutting head’s drive engine torque. However, the vibrations of the forces in both back supports have the same character as the course of the cutting head’s drive engine torque (a momentary torque increase causes a momentary load on the back supports).

Acknowledgements

The work has been implemented under the research project titled “Control of roadheader cutting heads movement for reduction of energy consumption of mining and dynamic loads” co-financed by the National Center for Research and Development under the Applied Research Projects (agreement no. PBS3/B2/15/2015).

References


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